

# IMVAC

INTERNATIONAL  
MACHINE VIBRATION ANALYSIS  
CONFERENCE

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## Low Speed Trunnion Bearing Problems

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# Low Speed Vibration Monitoring

## ANIMAL FEED DRYER PLANT



### Case Study 1 - **Trunnion Bearing Problem**

Gearbox Driven Drum Drive System

# Dryer System Overview

- Two Dryers on site A & B units.
- Critical to plant operation
- Removes excess water from high value DDGS product
- Reduced plant production rates if one dryer is offline

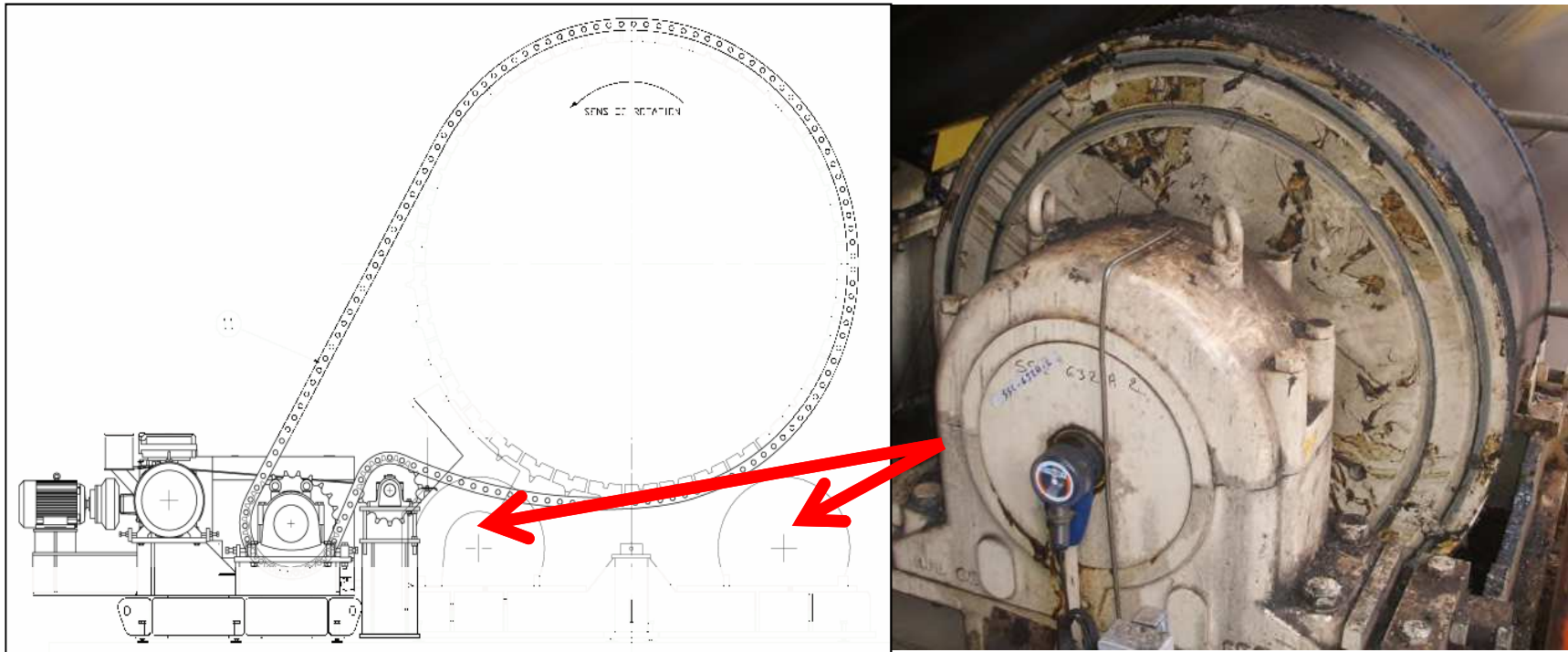


# Drive System Overview

- Typical production speeds for each dryer:
- Inverter Driven Drive Motor (1375 RPM)
- Gearbox Output (12.7 RPM)
- Chain Pinion Support Bearings (12.7 RPM)
- Trunnion Support Bearings (9.5 RPM)
- Main Dryer Drum (2.6 RPM)

# Drive System Overview

- 4 Supporting Trunions (8 Bearings) North/South





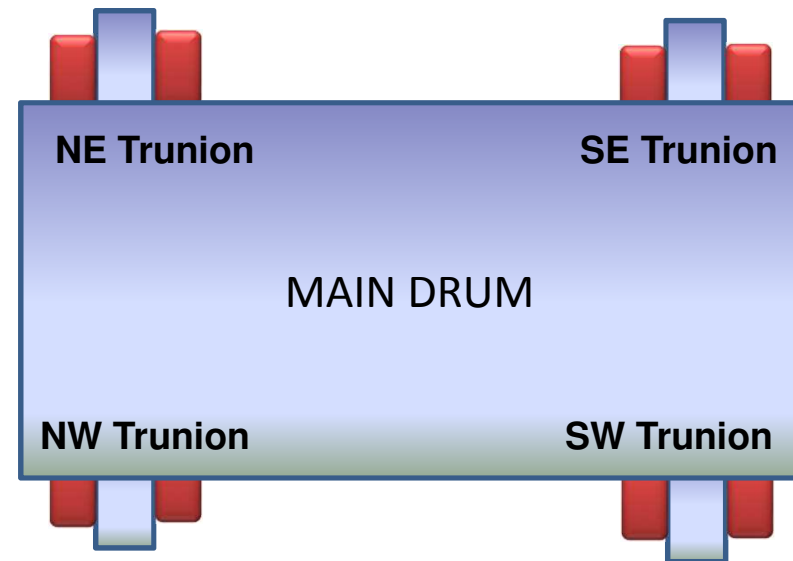
# Condition Monitoring

- Drum Drive System set-up for routine condition monitoring since plant start-up.
- Monthly Vibration Analysis on Motor, Gearbox, & Trunnion Bearings.
- 3 Monthly Oil Analysis on gearbox.



# Trunnion Bearing Monitoring

- Difficult to monitor using standard vibration techniques due to slow speed (9.5 RPM)
- **PeakVue Stress Wave** technology used.
- Very sensitive to impacting type faults such as **bearing defects, looseness problems & gear faults.**



# Trunnion Bearing Monitoring Slow Speed Setup

- Setup base around **PeakVue technology**
- Data collected in Acceleration
- To determine FMAX find highest defect freq of the bearing in orders of turning speed multiply by 8 (harmonics of the bearing defect freq)
- SKF 23164/CCW

Cage Freq	0.43
Roller Spin Defect Freq	3.95
Outer Race Defect Freq	9.65
<b>Inner Race Defect Freq</b>	<b>12.34</b>

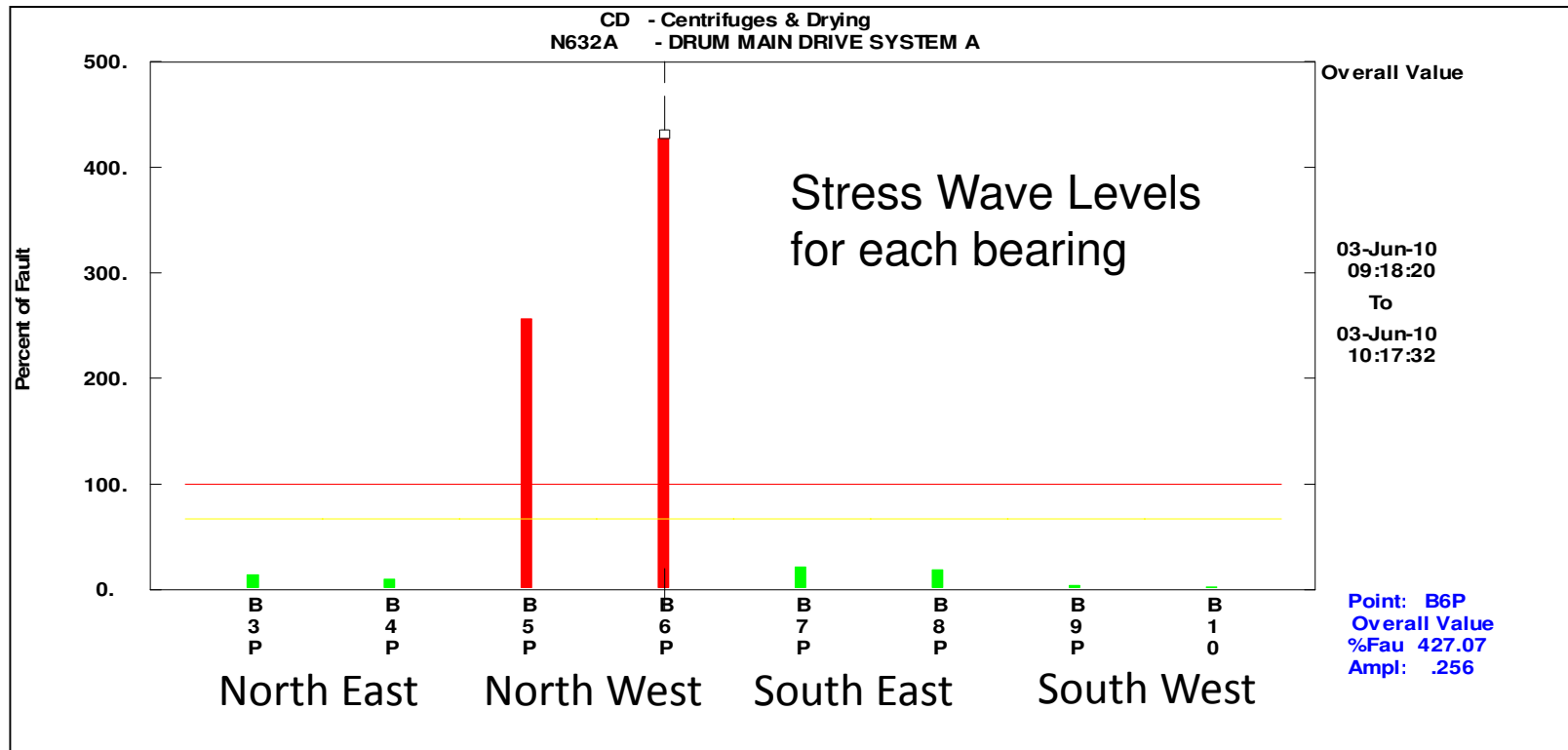


# Trunnion Bearing Monitoring Slow Speed Setup

- Set FMAX to 8 x highest bearing defect freq :
- **8 x 12.32** (Inner Race Defect Freq) = **98.4 orders**
- So FMAX in Cycles per min = Fmax x shaft RPM
- **98.4 X 9.5 RPM = 934 Cycles Per Min FMAX**
- Analyser will select closest band to this range
- **PeakVue** set to **500 Hz HP Filter**
- Capture at least **10 Revs** on the time waveform, min **4096** points on waveform.
- Lines of resolution **1600**
- Averages **1**

# Trunnion Bearing Analysis (PeakVue)

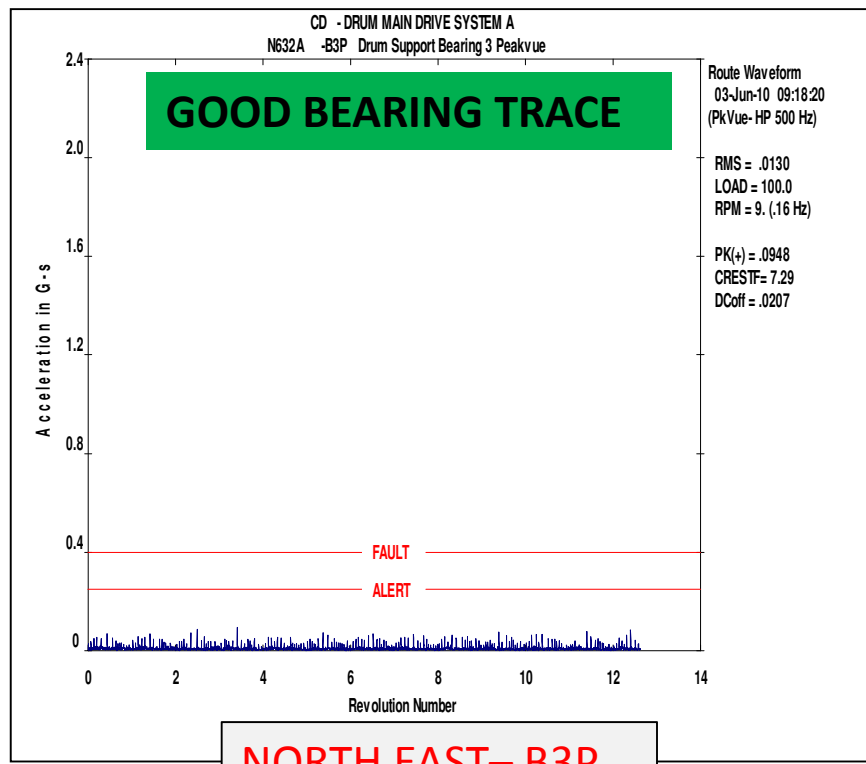
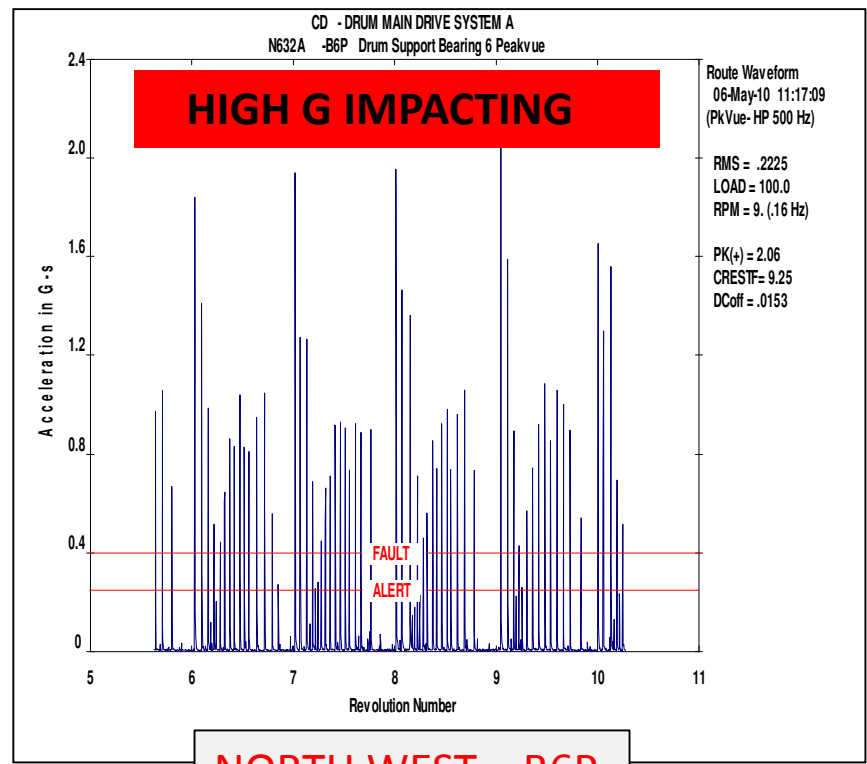
- PeakVue Overall Stress Levels in G's
- **Data showed raised impacting levels on North West Bearings.**





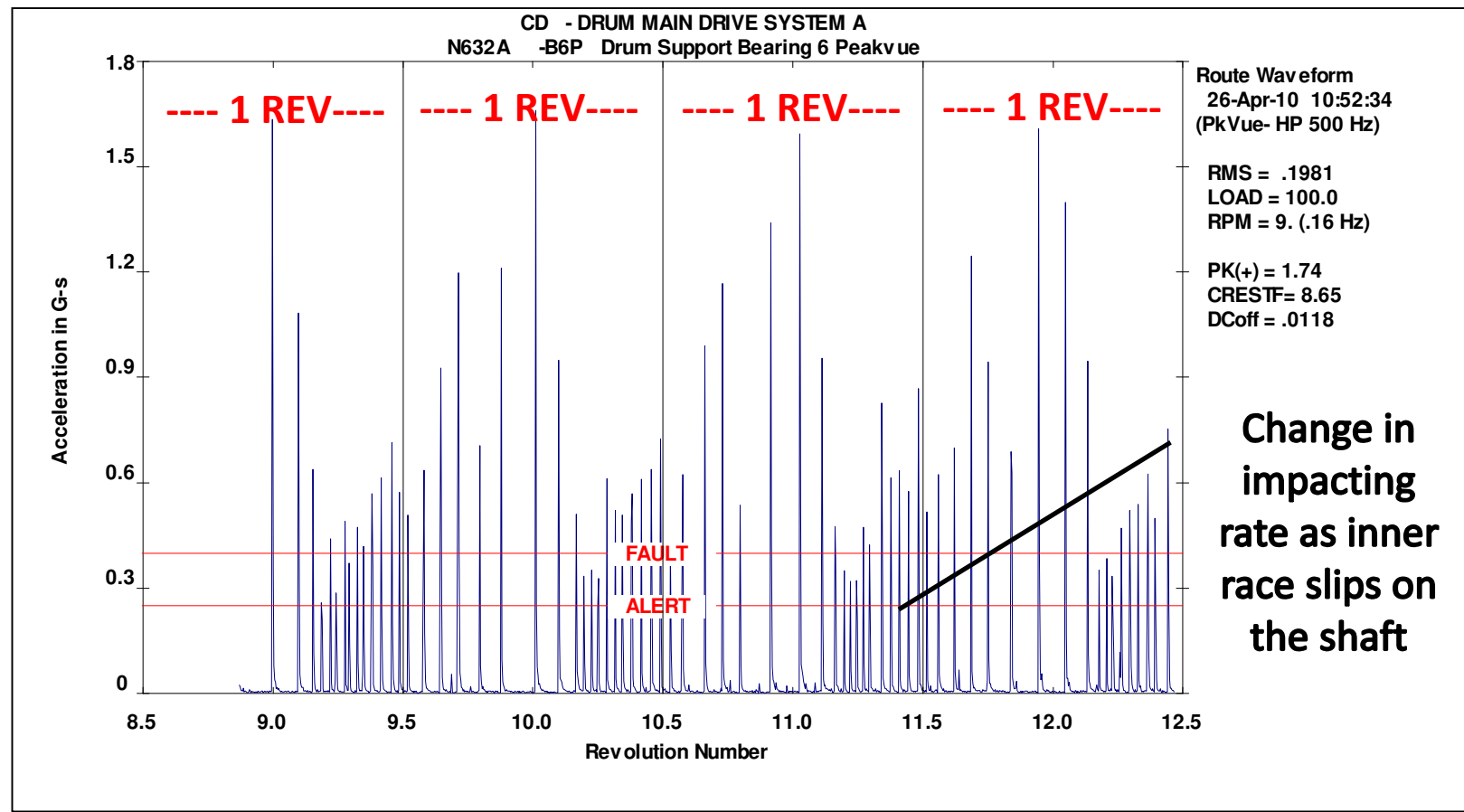
# Trunnion Bearing Analysis (PeakVue)

- **Comparison Time Waveforms, High G Stress Wave** impacting on the North West Bearing Pair



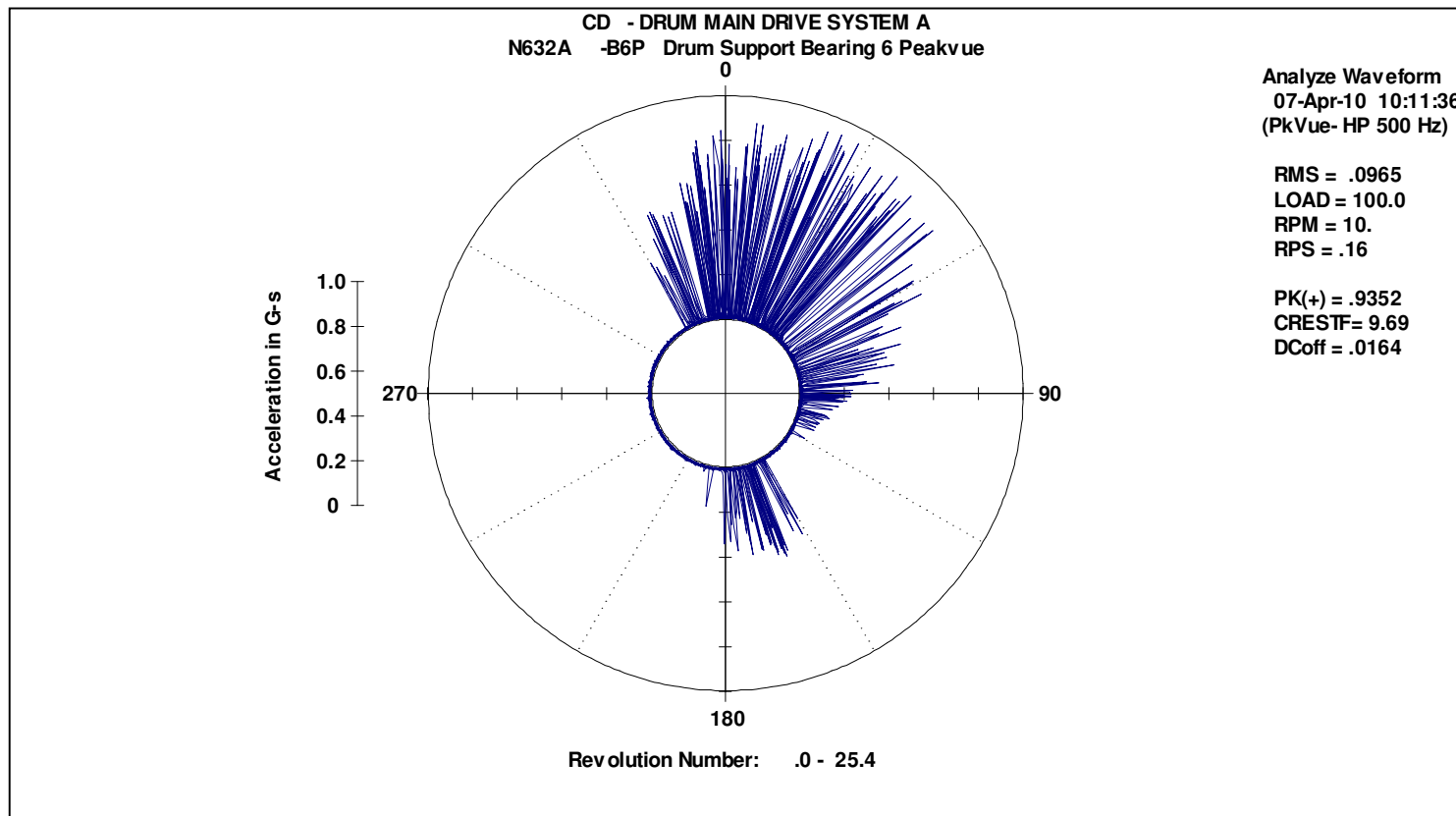
# Trunnion Bearing Analysis (PeakVue)

- B6P North West showed highest levels.
- Data pointed to an **INNER RACE PROBLEM** as impacts were modulated at 1XRPM (Due to rotation of inner race) many harmonics from 9.5 CPM (Shaft turning speed)



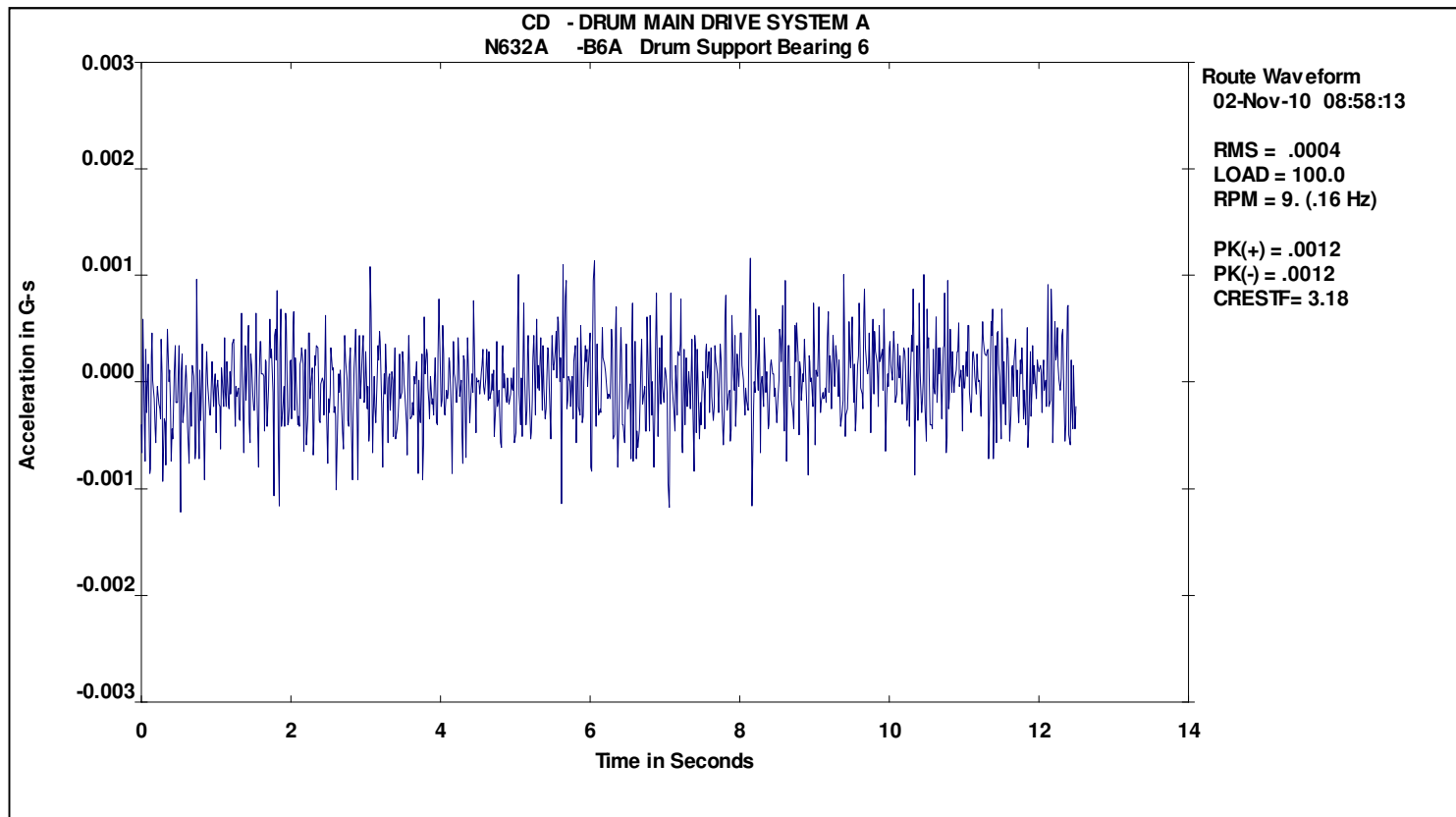
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# Trunnion Bearing Analysis (Standard)

- B6P North West Standard Data
- Data showed no impacting evident

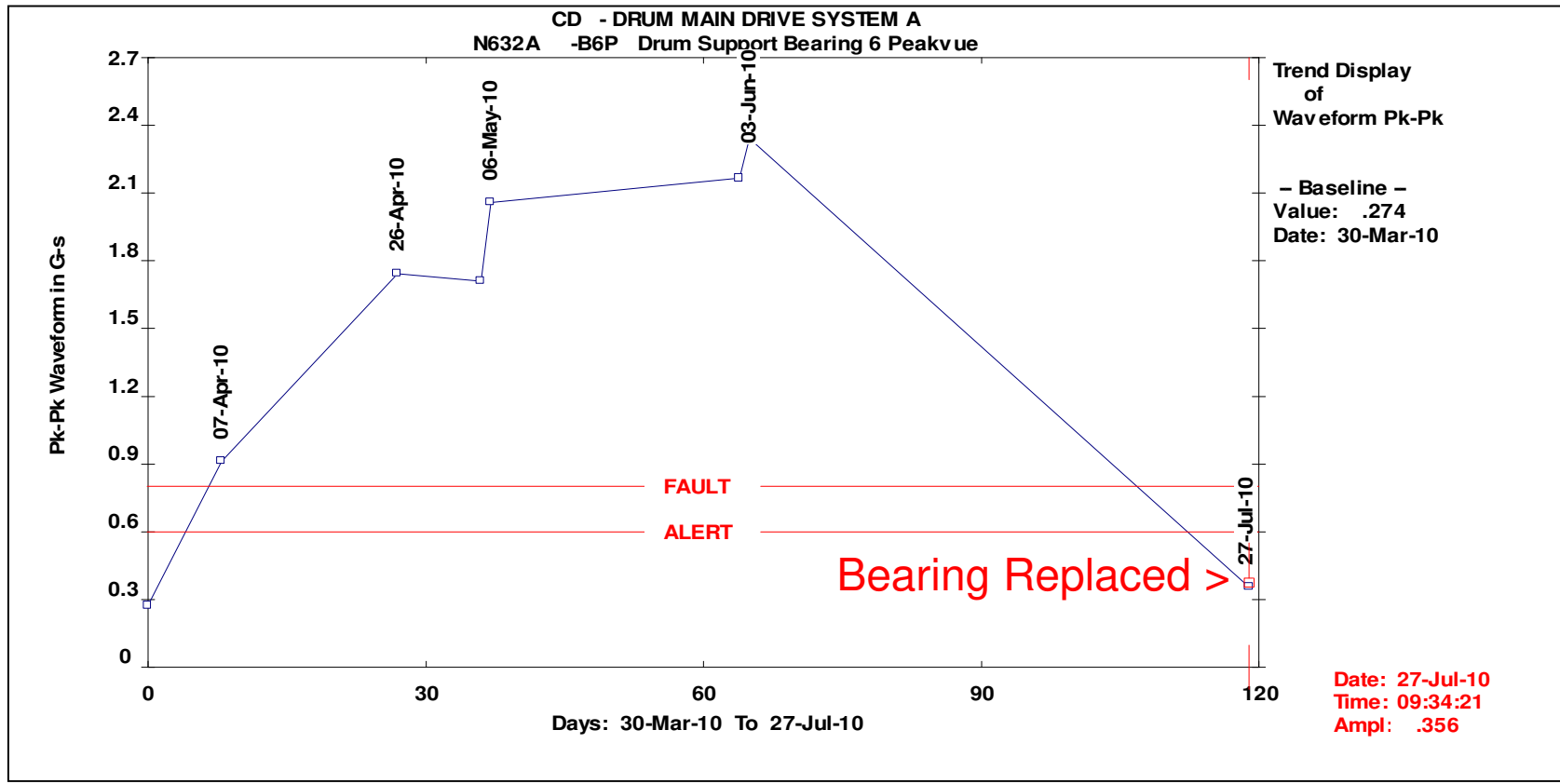






# Trunnion Bearing Analysis (PeakVue)

- **PeakVue Waveform PK-PK Trend** showed increasing levels over a 3 month period, 0.3 G's to 2.3 G's **RMS recommended North West Trunion replacement at July Shutdown**



# Dryer A – N-W Roller Bearing Pictures



The inner race bearing track shows some slight staining but no surface damage could be identified.

The outer race inside diameter shows some indentation damage. The vertical lines on the surface match the roller spacing.

The cage needs to be cleaned to allow more detailed analysis. The grease is much darker in colour than the new grease and is contaminated with brass particles.



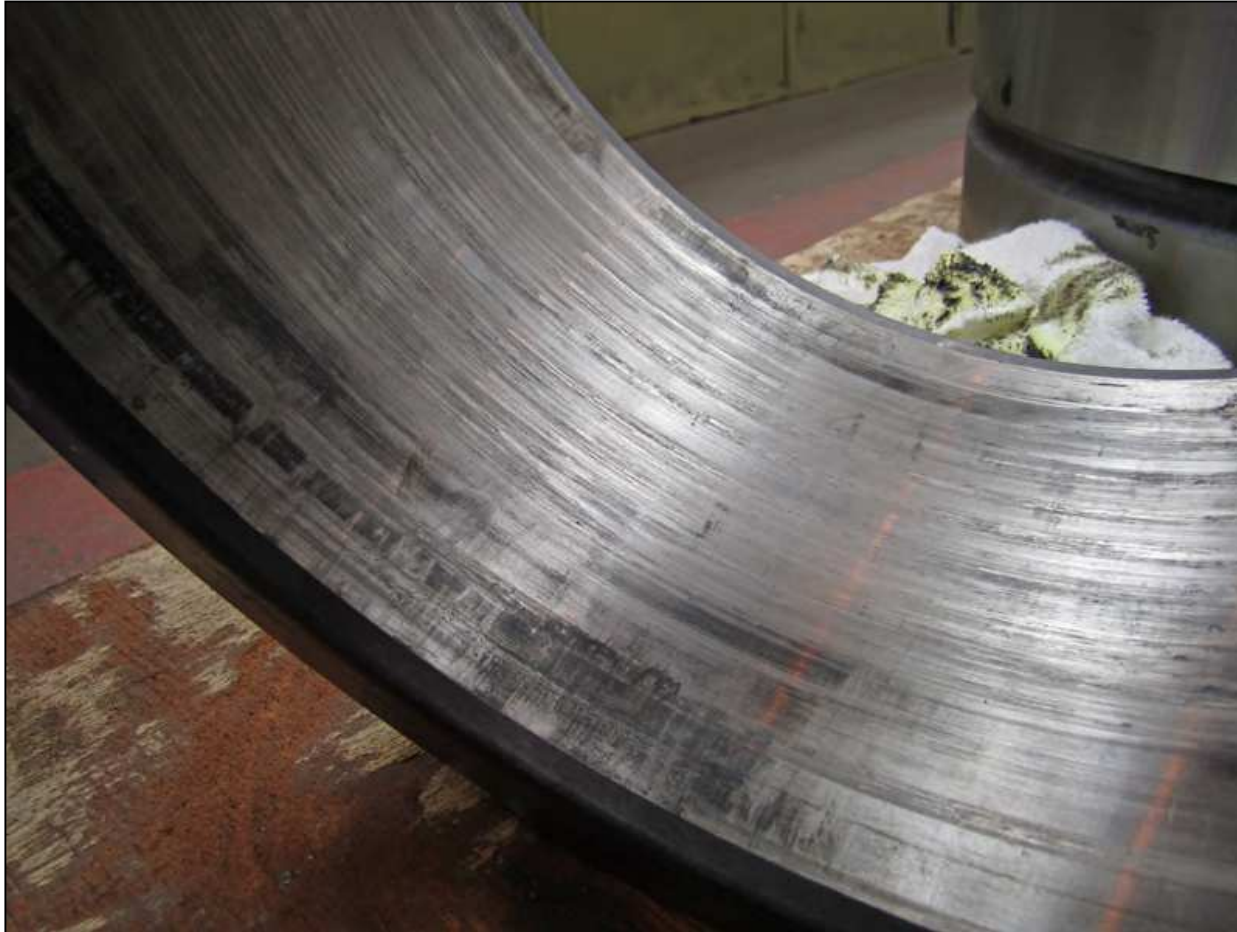
# Dryer A – N-W Roller Bearing Pictures

- **Root Cause – Poor inference between inner race and shaft**



**Roller Shaft –  
Horizontal scratches  
occurred during the  
removal of the bearing.  
The Damage around  
the surface of the  
bearing contact area is  
due to the bearing  
inner race rotating on  
the shaft.**

# Dryer A – N-W Roller Bearing Pictures

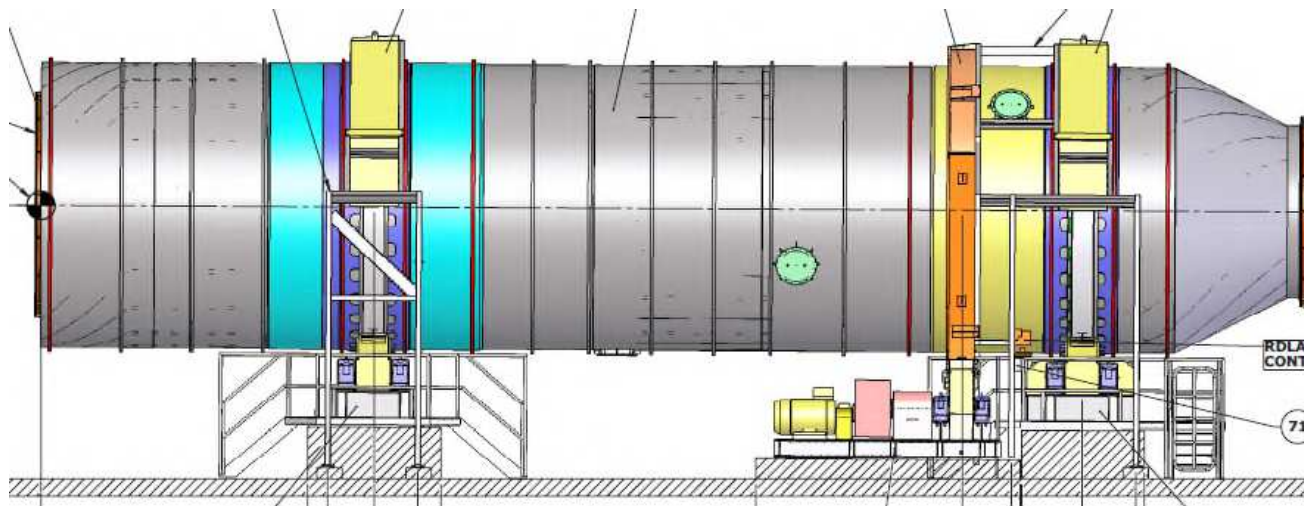


The inside diameter of the bearing inner race shows rotational damage



# Low Speed Vibration Monitoring

## ANIMAL FEED DRYER PLANT

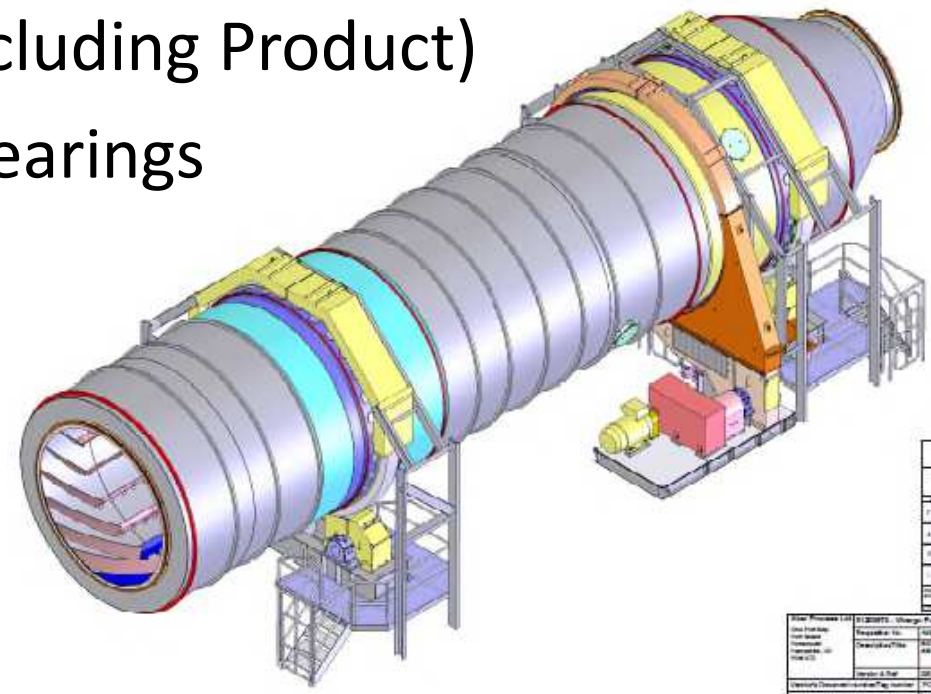


### Case Study 2 - **Trunnion Bearing Defect**

Gearbox Driven Drum Drive System

# Dryer Layout

- New Machine first run 2012
- Motor/Gearbox/Chain drive system
- 317te Rotating Drum (Including Product)
- 8 Supporting Trunnion Bearings
- 2 Thrust roller bearings
- Slow Speed





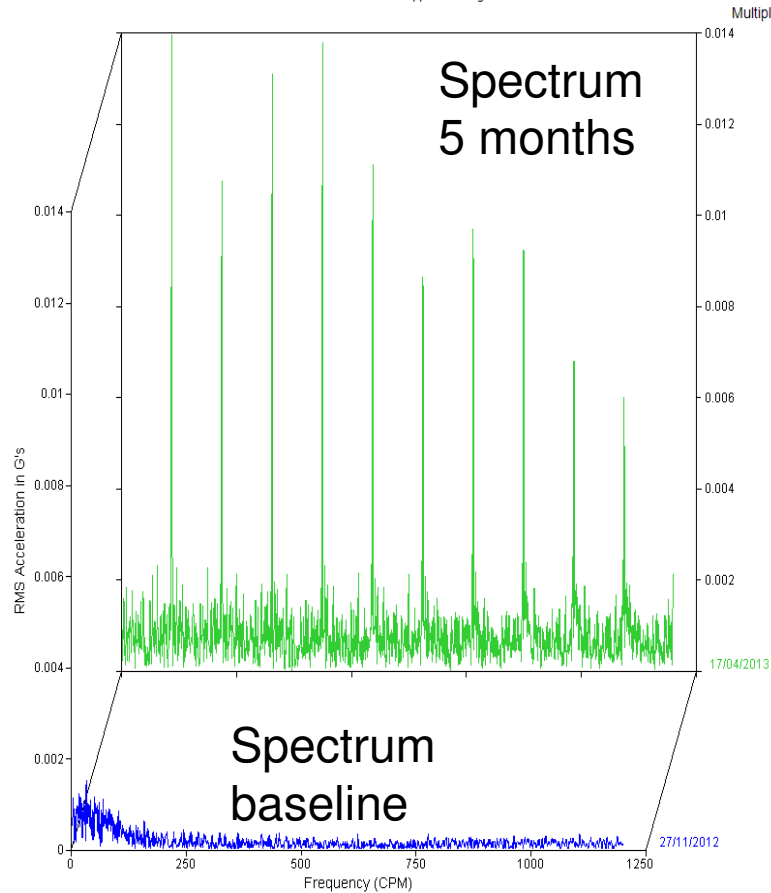
# Dryer Setup



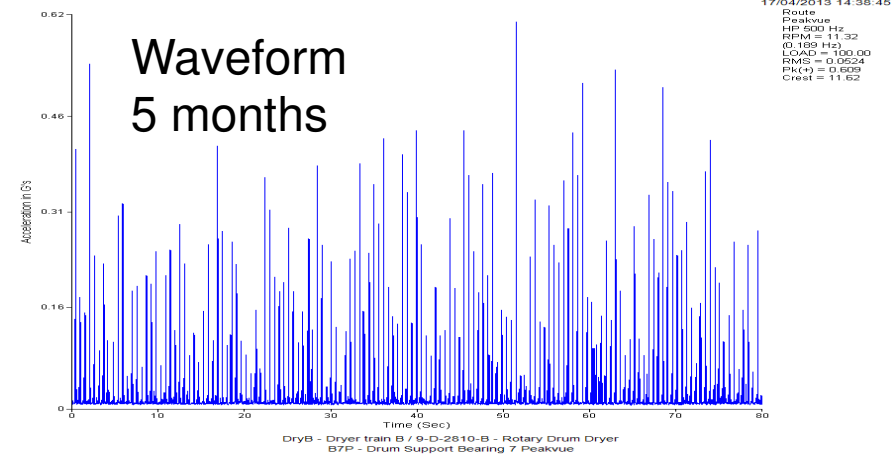
- Slow speed 11.3 RPM Trunnion bearing.
- PeakVue stress wave technology utilized
- PM setup for monthly readings

# Dryer – Vibration Analysis

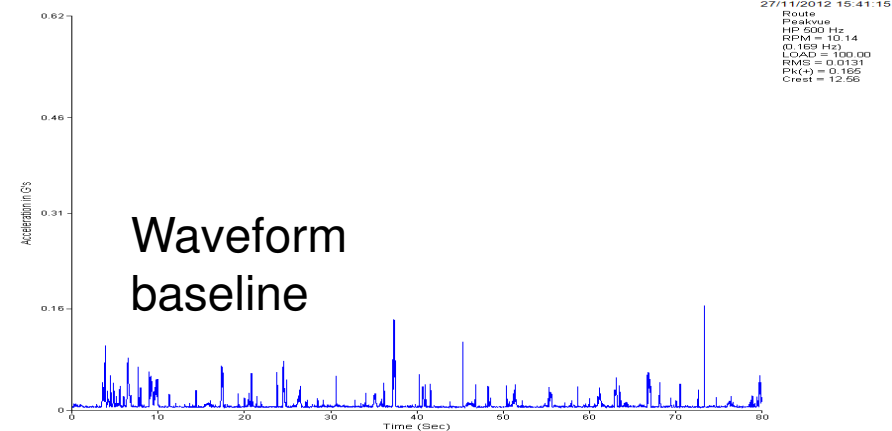
DryB - Dryer train B / 9-D-2810-B - Rotary Drum Dryer  
B7P - Drum Support Bearing 7 Peakvue



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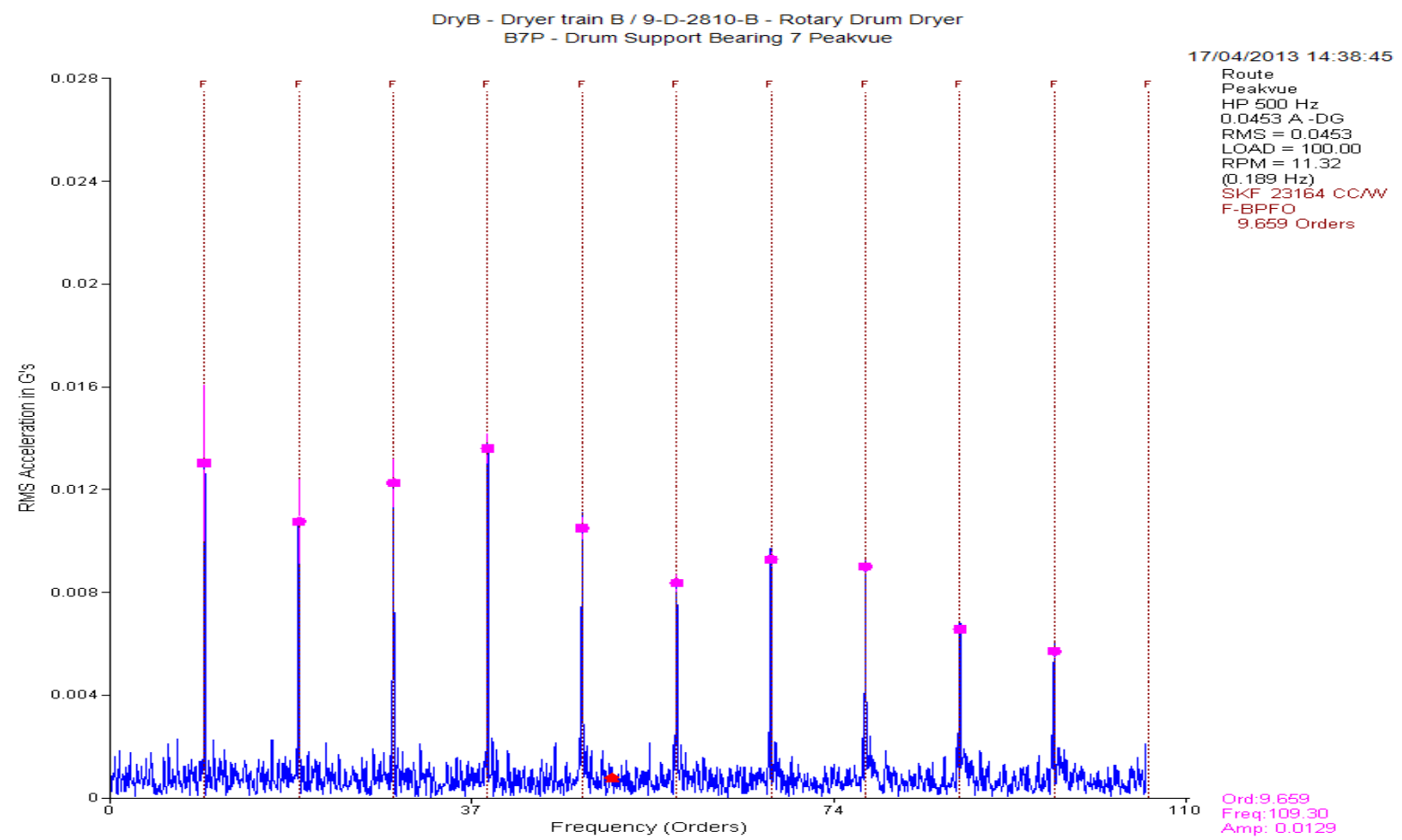
DryB - Dryer train B / 9-D-2810-B - Rotary Drum Dryer  
B7P - Drum Support Bearing 7 Peakvue



- Increase in spectral & waveform activity after 5 months of running on **B7P** bearing

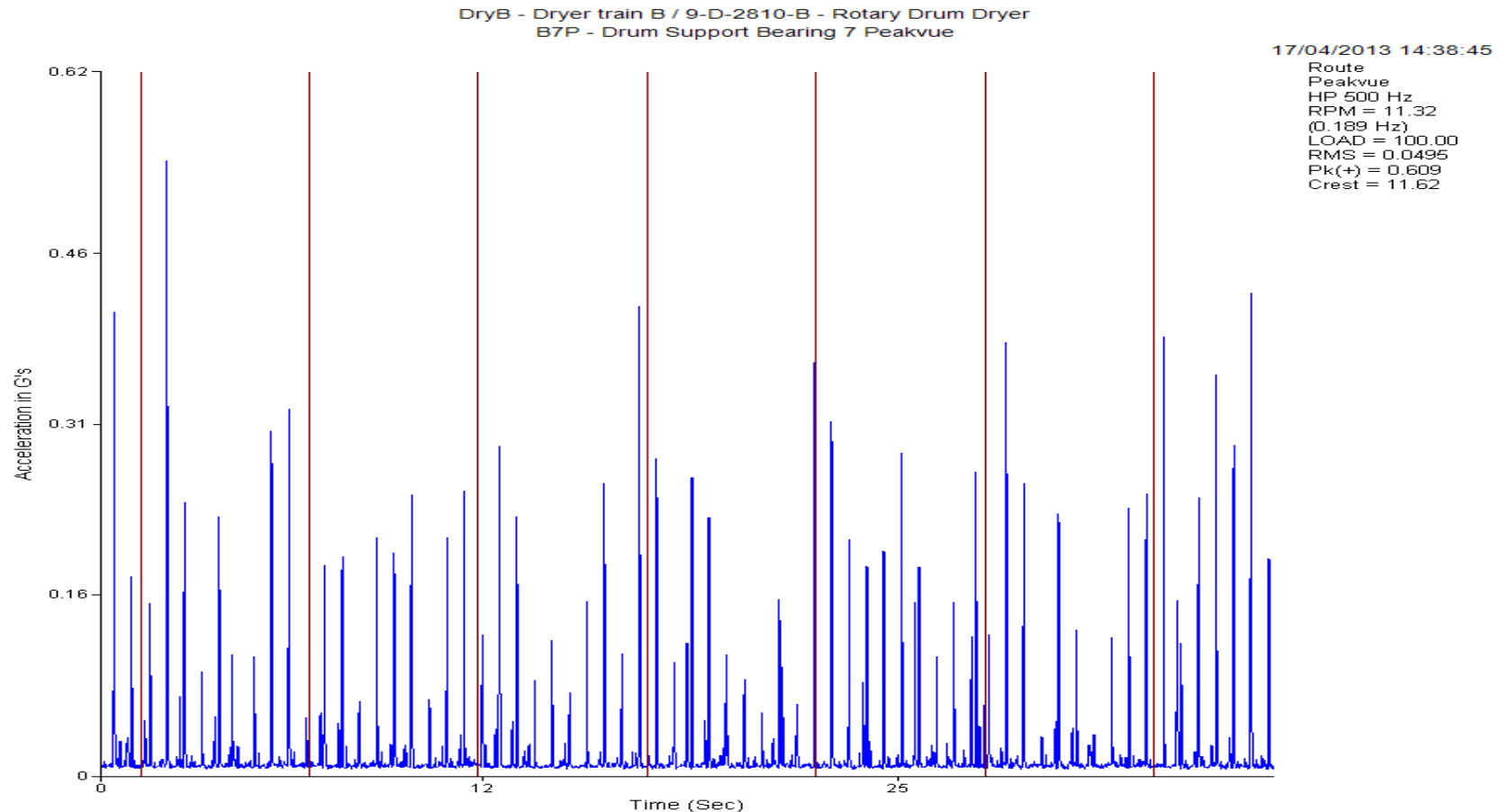


# Dryer – Vibration Analysis - B7P Brg



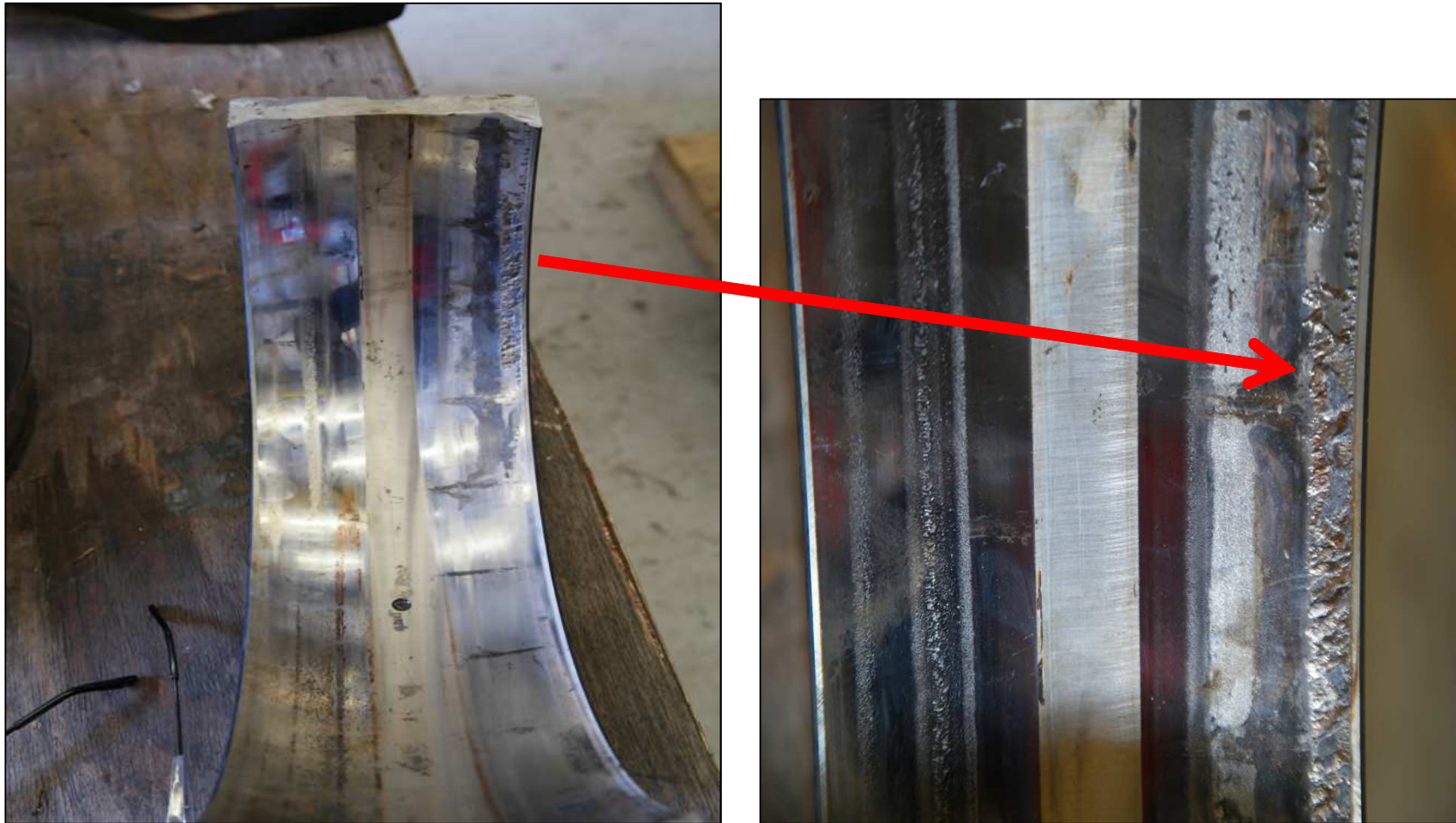
- Spectral data shows perfect match with **outer race defect freq for SKF 23164 CC/W (9.65 Orders)**

# Dryer – Vibration Analysis - B7P Brg



- Time waveform data shows impacts spaced at outer race defect frequency raised G levels.

# Dryer –B7P Bearing Inspection



- Inspection found advanced spalling on the outer race of the SKF 23164 CC/W bearing

# Dryer –B7P Bearing Damage Root Cause

- Bearing was sent away for detailed analysis
- Bearing damage was caused by **Excessive axial loading**
- Further investigations found **Incorrect alignment** of the trunnion bearings

